



- 1 Hotplate Precipitation Gauge Calibrations and Field Measurements
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## 11 Abstract –

12 First introduced in 2003, approximately 70 Yankee Environmental Systems (YES) 13 hotplate precipitation gauges have been purchased by researchers and operational meteorologists. A version of the YES hotplate is described in Rasmussen et al. (2011; R11). Presented here is 14 15 indoor- and field-based testing of a newer version of the hotplate; this device is equipped with longwave and shortwave radiation sensors. Hotplate surface temperature, coefficients describing 16 17 natural and forced convective sensible energy transfer, and radiative properties (longwave emissivity and shortwave reflectance) are reported for two of the new-version YES hotplates. 18 These parameters are applied in a new algorithm and used to derive liquid-equivalent 19 20 accumulations for snowfall and rainfall; these accumulations are compared to values derived by the internal algorithm used in the YES hotplates (hotplate-derived accumulations) and to 21 weighing gauge accumulations. In contrast with R11, the new algorithm accounts for radiative 22 terms in a hotplate's energy budget, applies an energy conversion factor which does not differ 23 from a theoretical energy conversion factor, and applies a surface area that is correct for the YES 24 hotplate. Radiative effects are shown to be relatively unimportant for the precipitation events 25 analyzed. In addition, this work documents a 10 % difference between the hotplate-derived and 26 new-algorithm-derived accumulations. This difference seems consistent with R11's application 27 28 of a hotplate surface area that deviates from the actual surface area of the YES hotplate and with 29 R11's recommendation for an energy conversion factor that differs from that calculated using thermodynamic theory. 30

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# 32 **1 - Introduction**

33	Two types of instrumentation have been developed for measuring liquid-equivalent
34	snowfall rates and liquid-equivalent snow accumulations: 1) Weighing gauges that measure
35	snowfall as it collects in a container or on a surface (WMO, 2008), and 2) optical gauges that
36	measure the concentration and size of snow particles either in free fall or within a wind tunnel
37	(Loffler-Mang and Joss, 2000; Deshler, 1988). Many of these gauges obstruct the wind and thus
38	cause falling snow particles to deflect from the measurement zone. Consequently, rates and
39	accumulations are underestimated and should be adjusted to account for undercatch (Jevons,
40	1861; Lovblad et al., 1993). Alternatively, both gauge types can be operated within a fenced
41	enclosure that minimizes wind and the resultant undercatch (Goodison et al., 1998; Rasmussen et
42	al., 2012). In addition, optical gauges require a snow particle density to convert concentration
43	and size to a liquid-equivalent rate and accumulation (Brandes et al., 2007; Lempio et al., 2007).
44	Because this density is variable and difficult to measure accurately (Locatelli and Hobbs, 1974),
45	optical snowfall measurements are uncertain and remain uncertain even if undercatch is
46	accounted for. A further disadvantage, for both the weighing and optical devices, is that the
47	entrance to the device can become clogged with snow (Warnick, 1954; Currie, 1998; Stickel et
48	al., 2005).
49	The Yankee Environmental Systems (YES, 2011) hotplate was developed to minimize
50	the aforementioned uncertainties. Advantages of the hotplate are: 1) it is compact, 2) it is
51	immune to clogging, 3) there is no requirement that snow particles fall through an opening, and
52	4) the derived rates and accumulations are largely independent of snow particle density, although

53 a dependence does exist (R11; their figure 14).





This work furthers efforts to advance the hotplate as a snowfall measurement system 54 55 (Borkhuu, 2009; R11; Boudala et al., 2014). We develop calibration constants for two hotplate systems configured with longwave and shortwave radiation sensors. These are a hotplate gauge 56 owned by the University of Wyoming (UW) and a hotplate gauge owned by the National Center 57 for Atmospheric Research (NCAR; Boulder, CO). In addition, we develop a new hotplate data 58 processing algorithm, derive liquid-equivalent rates and accumulations for 27 precipitation 59 events (snowfall and rainfall), compare accumulations obtained with the new algorithm to those 60 derived by an internal algorithm (hotplate-derived accumulations), and compare accumulations 61 to values derived using weighing gauges. 62

#### 63 2 - Algorithm Development

The two vertically-stacked circular aluminum plates seen in Fig. 1 are the precipitation 64 measurement portion of the YES hotplate system. The plate diameter  $(D_h)$  is 0.130 m and both 65 plates have concentric rings that extend vertically either 3 mm (inner and middle rings) or 1 mm 66 (outer ring) from the plate surface. One of the plates faces upward and is exposed to 67 precipitation, the other faces downward. Temperature sensors monitor the top and bottom plates 68 and feedback-controlled heaters maintain the plates at approximately 75 °C (R11). Electrical 69 power supplied to the top plate  $(Q_{top})$  compensates for power lost via sensible, net radiative, and 70 latent (vapor mass) transfer. The power input to the bottom plate  $(Q_{bot})$  is the source term in that 71 72 plate's energy budget and is assumed to only compensate for sensible power output. This 73 compensation is the basis for the hotplate's determination of wind speed (U). Values of U are commonly used to evaluate a Reynolds number (Re). The Reynolds number controls sensible 74 heat transfer from a ventilated surface (Kobus and Wedekind, 1995). A complete description of 75 76 our nomenclature is provided in the Appendix.





78 79	Since the hotplate was introduced in 2003	3, two teams (Borkhuu, 2009; R11) ha	ive
80	reported data processing algorithms. The algorith	nm in Borkhuu (2009) can be explaine	ed by
81	reference to the equation she used to model the te	op plate's power budget:	
82	0 =	Implied Steady-state	
83	$Q_{top}$	Electrical Power Supplied to Top	Plate
84	$-D_h\cdot K_x\cdot (T_h-T)\cdot (\gamma+\alpha\cdot Re^\beta)$	Sensible Power Output	
85	- $P \cdot E / f_2$	Latent Power Output	(1)
86	In Eq. 1, there are three terms that sum to zero in	an assumed steady-state. The last of	these, the
87	latent power output, is proportional to the precipi	tation rate (P) and a snow particle cat	tch
88	efficiency ( <i>E</i> ) and inversely proportional to $f_2$ , are	electrical-to-precipitation conversion	n factor. In
89	addition, the sensible power output term has con-	ributions from natural convection (pr	oportional
90	to $\gamma$ ) and forced convection (proportional to $\alpha \cdot R$	$e^{\beta}$ ), where $\alpha$ , $\beta$ , and $\gamma$ are fitted constant	ants. These
91	convective regimes are discussed in Kobus and V	Vedekind (1995) and are shown graph	nically in
92	their Figure 6. Eq. 1 is similar to the algorithm u	sed by King et al. (1978) to derive clo	ud liquid
93	water concentration using a heated airborne sens	or.	
94	The algorithm in R11 is based on Eq. 2.		
95	$P = [Q_{top} - Q_{bot} - f_I(U)] \cdot f_2 / E$		(2)
96	Here, $f_l(U)$ is a wind speed-dependent function.	Also in Eq. 2, we see the conversion f	actor
97	introduced in the previous paragraph. Somewhat	different from how R11 formulated th	heir
98	conversion factors for rain and snow, we formula	the $f_2$ to account for the warming of ice	e, melting,
99	warming of the liquid, and liquid evaporation. For	or rain, we formulate $f_2$ to account for	the
100	warming of liquid and its evaporation. With an e	xception that we justify later, we appl	ied the





101 0	conversion factors as reco	mmended by R11: 1) if 7	$\Gamma < 0$ °C, the snow $f_2$	is applied, and	2) if $T > 4$
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102 °C the rain  $f_2$  is applied.

103	In Eq. 1, the sensible term is a function of	<i>Re</i> , and thus <i>U</i> , and also a function of <i>T</i> .	
104	Hence, Eq. 1 can be rearranged to look similar to Eq. 2 with P dependent on T, U, $Q_{top}$ , $f_2$ , and		
105	A difference between the Eq. 1 and Eq. 2 formula	tions is the explicit dependence on $Q_{bot}$ , in Eq.	
106	2; this is in addition to the implicit $Q_{bot}$ -dependent	wind speed in $Re$ (Eq. 1) and in $f_l(U)$ (Eq. 2).	
107	Borkhuu (2009), YES (2011), and R11 surmised that the energetic effect of longwave		
108	and shortwave radiation could, in some settings, b	e comparable to the latent power term.	
109	Consequently, our hotplate (Wolfe and Snider, 20	12) was upgraded by YES in 2011. The	
110	upgrade included radiation sensors for the measur	ement of longwave and shortwave fluxes. An	
111	objective of this paper is the incorporation of the n	adiation measurements into a new precipitation	
112	rate algorithm.		
113	The following budget equation is the basis	for our analysis:	
114	0 =	Implied Steady-state	
115	$Q_{top}$	Electrical Power Supplied to Top Plate	
116	$-D_h\cdot K_x\cdot (T_h-T)\cdot (\gamma+\alpha\cdot Re^\beta)$	Sensible Power Output	
117	- $A_h \cdot \varepsilon_h \cdot \sigma \cdot T_h^4$	Longwave Power Output	
118	$+A_h\cdot \varepsilon_h\cdot IR_d$	Longwave Power Input	
119	$+A_h \cdot (I-R_h) \cdot SW$	Shortwave Power Input	
120	- $P \cdot E / f_2$	Latent Power Output (3)	
121	Compared to Eq. 1, Eq. 3 has three additional term	ns. These describe the interaction of the	

122 hotplate with its environment via radiative transfer. Two of these terms are inputs (longwave and

123 shortwave) and one is an output (longwave).





#### 124 2.1 - Hotplate Data Files

The hotplate outputs data to two files. The previously discussed  $Q_{top}$  and  $Q_{bot}$  are two of several recorded variables and both of these are essential for the analysis described here. One of the files is known as the UHP or "user" hotplate file. The UHP file is provided to all YES customers. The second file is the SHP or "sensor" file. Table 1 has the list of all recorded variables and how some of these are symbolized. A complete list of variables (measured and computed), and constants, is provided in the Appendix. With the exception of Unix time, all variables in Table 1 are available as 60-s running averages, sampled at 1 Hz (YES, 2011).

#### 132 **2.2 - Radiative Properties**

Two radiative properties are applied in this analysis. In the infrared, or longwave, the 133 emissivity of the hotplate is the relevant property. The material used to fabricate the plates is 134 135 aluminum, which when exposed to air becomes covered with an aluminum oxide layer. Hence, 136 the hotplate emissivity was taken to be that of oxidized aluminum. The value we picked is  $\varepsilon_{h}$ 0.14 (Weast, 1975; Section E). Furthermore, we made two assumptions: 1) the longwave output 137 (Eq. 3) is the product of  $\mathcal{E}_h$  (assumed constant), hotplate area ( $A_h$ ), and the flux emitted by a black 138 body at  $T_h$ , and 2) the longwave input (Eq. 3) is the product  $\varepsilon_h$ ,  $A_h$  and the downwelling 139 longwave flux ( $IR_d$ ). In a later section, we explain how we derive  $IR_d$ . 140 In the visible, or shortwave, the hotplate's reflectance  $(R_h)$  is the relevant property. Eq. 3 141 142 shows how we factored into a hotplate's energy budget the reflectance, a measured shortwave flux (SW; Table 1), and  $A_h$ . A value for  $R_h$  was determined as follows. We exposed the UW 143 hotplate to solar illumination, while measuring the solar flux, and then shaded the hotplate to 144 145 establish a baseline for the determination of  $R_h$ . During these experiments, there was negligible wind and therefore natural convection dominated forced convection in the budget. The energy 146





- budget equation we used to analyze these measurements has three terms:  $Q_{top}$ , sensible power output, and solar input. In two experiments, the values  $R_h = 0.66$  and  $R_h = 0.61$  were derived. We apply the average of these ( $R_h = 0.63$ ) in our analysis of measurements from both UW and NCAR hotplates. Because of the oxide layer, the derived reflectance is smaller than the value reported for polished aluminum reflecting "incandescent" light (0.69; Weast, 1975; Section E) and significantly smaller than the value for vacuum-deposited aluminum at visible wavelengths (0.97; Hass, 1955).
- 154 **3 Methods**

#### 155 3.1 - Site Description

156 Indoor testing was conducted in a high-bay weather balloon hangar and in a laboratory. These facilities are at the University of Wyoming (UW) and are abbreviated hangar and lab. 157 During wintertime, and especially at night, the hangar is cold (~ 0 °C); the lab is warm year 158 round (~ 20 °C). Field measurements (Table 2) were conducted in Southeast Wyoming at the 159 160 Glacier Lakes Ecosystem Experiments Site (GLE), in Southeast Wyoming near the summit of Battle Pass (BTL), and at the North Redfield site in Western New York (OWL). During both 161 162 indoor and field measurements, all parameters reported by the hotplate (UHP and SHP variables; 163 section 2.1) were recorded using a custom-built data system. 164 The accuracy of a hotplate-estimated precipitation rate is dependent on assessment of 165 whether the sensed hydrometeors are rain or snow (R11 and Fig. 3a). We infer the latter using a 166 calculated ice-bulb temperature  $(T_{IB})$  (Iribarne and Godson, 1981; Chapter 7). Our basis for the 167  $T_{IBS}$  are measurements of relative humidity (RH; 100 % when saturated with respect to liquid),

168 temperature, and pressure (Table 1). The lower limits on the  $T_{IB}$ s, assuming the measured RH is





- 169 overestimated by 5 % (YES, 2011), is no more than 0.4 °C colder than the values we report. In
- instances with lower-limit  $T_{IB}$ s larger than 0 °C, we assume the sensed hydrometeors were liquid.

## 171 **3.2 - NOAH-II Gauge**

- 172 The NOAH-II is a weighing-type gauge manufactured by ETI Instrument Systems Inc.
- 173 (www.etisensors.com). NCAR operated a NOAH-II at GLE and BTL during 2012, and coauthors
- 174 (Campbell and Steenburgh) operated a NOAH-II at OWL (Dec. 2013 through Jan. 2014;
- 175 Campbell et al., 2016). The three NOAH-II gauges were outfitted with Alter shields (Goodison
- 176 et al, 1998; hereafter G98).

## 177 **3.3 – Indoor Testing**

178 Indoor testing of the UW hotplate was conducted every year from 2011 to 2015; the

179 NCAR hotplate was only tested in 2012. Based on our testing of the UW hotplate, we have no

180 evidence indicating that the calibration changed over the duration of any of the field

181 deployments; however, Wettlaufer (2013) does demonstrate that calibration constants did change

182 over the 2011 to 2015 interval in response to servicing conducted twice at YES. In this paper we

183 present and apply calibration constants appropriate for the UW hotplate sensor deployed at GLE

184 (April 2012) and at OWL (December 2013 through January 2014).

185 During testing, we controlled the hotplate's radiation environment by placing a material

186 with known emissivity (painted-steel sheeting,  $\varepsilon_s = 0.84$ ) above and below the hotplate. The steel

187 sheets were positioned to dominate the hotplate's upward and downward fields of view (Fig. 2);

188 however, the sheets were positioned vertically so that they were not heated by the hotplate. In

189 that case, the sheet temperature  $(T_s)$  can be assumed equal to T.





191 3.4 - Downwelling Longwave Flux As we have already mentioned, previous work concluded that the hotplate method of 192 193 determining precipitation amount can be affected by longwave radiation. In response to that 194 finding, YES incorporated a device that measures the net downward longwave flux (pyrgeometer, e.g., Albrecht et al., 1974) into the system (Table 1). 195 196  $M_{IR} = IR_d - IR_u$ (4)The left-hand side of Eq. 4 represents the longwave radiant measurement  $(M_{IR})$  and the right-197 198 hand side has the downward and upward components contributing to  $M_{IR}$ . Because  $IR_d$  appears in the hotplate's budget (Eq. 3), and since  $M_{IR}$  is the only term in Eq. 199 4 that is measured, the upwelling component  $(IR_u)$  must be evaluated. This is possible because 200 201 the signal from the pyrgeometer is adjusted, within the hotplate electronics package, to make the source of the upwelling infrared flux a virtual blackbody at the ambient temperature (YES 2012, 202 203 personal communication). In that case,  $IR_d$  can be formulated as  $IR_d = M_{IR} + \sigma \cdot T^4$ 204 (5)205 where  $\sigma$  is the Stefan-Boltzmann constant and T is the hotplate-measured ambient temperature. 206 We also use Eq. 6 to calculate the downwelling infrared flux  $IR_d = \varepsilon_s \cdot \sigma \cdot T_s^4$ 207 (6)3.5 – Warm-Cold Ambient Temperature Tests 208 Procedures described here were applied during testing conducted indoors (hangar and lab, 209 210 section 3.1) at two different temperatures and are hereafter referred to as the warm/cold test. We

show how values of a warm  $(T_w)$  and cold  $(T_c)$  ambient temperature, combined with other

recorded hotplate variables (Table 1), can be used to derive two settings in Eq. 3 ( $T_h$  and  $\gamma$ ). In





our analysis, the temperature of the steel sheeting  $(T_s)$  is assumed equal to the ambient

temperature (either  $T_w$  or  $T_c$ ) and  $IR_d$  is calculated with Eq. 6. By design these tests had

215 negligible forced-convective and latent transfers. In that case, Eq. 7a - b are the budget

216 equations.

217 
$$0 = Q_{top,w} - D_h \cdot K_x \cdot (T_h - T_w) \cdot \gamma - A_h \cdot \varepsilon_h \cdot \sigma \cdot T_h^4 + A_h \cdot \varepsilon_h \cdot \varepsilon_s \cdot \sigma \cdot T_w^4 + A_h \cdot (1 - R_h) \cdot SW_w$$
(7a)

218 
$$0 = Q_{top,c} - D_h \cdot K_x \cdot (T_h - T_c) \cdot \gamma - A_h \cdot \varepsilon_h \cdot \sigma \cdot T_h^4 + A_h \cdot \varepsilon_h \cdot \varepsilon_s \cdot \sigma \cdot T_c^4 + A_h \cdot (1 - R_h) \cdot SW_c$$
(7b)

219 The measurements applied in these equations were  $T_w$  and  $T_c$ , the warm and cold plate powers

220  $(Q_{top,w} \text{ and } Q_{top,c})$ , the warm and cold shortwave fluxes  $(SW_w \text{ and } SW_c)$ , and constants

221 (Appendix). Values of  $T_h$  and  $\gamma$  (hereafter referred to as  $T_h/\gamma$  pairs) were derived by minimizing

222 departures from zero simultaneously in Eq. 7a - b. Minimization was conducted using a

223 Newton's method equation solver (Exelis Visual Information Solutions, Inc.); the convergence

tolerance was  $1 \times 10^{-4}$  J s<sup>-1</sup>.

## 225 3.6 - Nusselt-Reynolds Relationship

The Nusselt number ( $Nu = \gamma + \alpha \cdot Re^{\beta}$ ), is a component of the sensible power output term in Eq. 3. In this section, we develop a relationship between Nu and Re based on measurements recorded in the field when precipitation was not occurring; in a later section we show how that relationship is applied in a calculation of the precipitation rate.

Conceptually, *Nu* is a dimensionless representation of the sensible power output. Eq. 8a was used to calculate *Nu* with measurements ( $Q_{top}$ , *T*, and *SW*), a calculated variable (*IR<sub>d</sub>*; section 3.4), and constants (Appendix and Table 3).

233 
$$Nu = [Q_{top} - A_h \cdot \varepsilon_h \cdot \sigma \cdot T_h^4 + A_h \cdot \varepsilon_h \cdot IR_d + A_h \cdot (1 - R_h) \cdot SW] / [D_h \cdot K_x \cdot (T_h - T)]$$
(8a)

In the denominator of Eq. 8a is a term proportional to the sensible power output due to molecularconduction.





236	Conceptually, Re is a dimensionless representation of the wind speed. Eq. 8b was used to

calculate Re with a measurement (U) and constants (Appendix).

238 
$$Re = p_x \cdot D_h \cdot U / (R_d \cdot T_x \cdot \mu_x)$$
(8b)

Two criteria were used to select a site-specific data subset for the *Nu-Re* development: 1)

no precipitation, and 2) at least three hours of continuous measurements with a broad range of

241 wind speeds. We fitted the selected Nu-Re pairs using a non-linear least squares procedure

242 (curvefit; Exelis Visual Information Solutions, Inc.); the convergence tolerance for the relative

243 decrease in chi-squared was  $1 \times 10^{-3}$ .

# 244 3.7 - Electrical-to-precipitation Conversion Factor

Equilibrium thermodynamics, with the assumptions that ice melts at  $T_o = 0$  °C and vaporization occurs at  $T_h$ , was used to derive the conversion factor in Eq. 3 ( $f_2$ ). Adopting the temperature criteria from R11 (also see section 2), and a framework from Iribarne and Godson (1981; Chapter 7), we formulated theoretical conversion factors as  $f_2(T, T_h) = \{\rho \cdot A_h \cdot [C_i \cdot (T_o - T) + L_f(T_o) + C \cdot (T_h - T_o) + L_v(T_h)]\}^{-1}$  (T < 0 °C) (9a)

250 
$$f_2(T, T_h) = \{ \rho \cdot A_h \cdot [C \cdot (T_h - T) + L_\nu(T_h)] \}^{-1} \qquad (T > 4 \text{ }^{\circ}\text{C}) \qquad (9b)$$

This formulation is graphed in Fig. 3a (solid line) where we extended Eq. 9b into the temperature range (0 °C < T < 4 °C) where the distinction between rain and snow is ambiguous because falling snow particles remain unmelted in situations with T > 0 °C and low humidity (R11).

- We now compare the conversion factor derived using Eq. 9a b with that reported in
- 255 R11. To be consistent with R11, we assume  $T = T_h = 0$  °C. We find that the ratio of  $f_2$  (Eq. 9a)
- divided by the factor reported in R11 for snow (3.99 x  $10^{-8}$  m J<sup>-1</sup>) and the ratio of  $f_2$  (Eq. 9b)
- divided by the factor reported in R11 for rain ( $4.52 \times 10^{-8} \text{ m J}^{-1}$ ), are both 0.666. Since these





258	ratios are equal to the area in R11 ( $A_h = 0.008844 \text{ m}^2$ ), divided by the area applied in our
259	calculation ( $A_h = (\pi/4) \cdot 0.130^2 = 0.01327 \text{ m}^2$ ), we conclude that the discrepancy is not due to
260	differing thermodynamic parameters applied in R11's and our calculations (e.g., the latent heat
261	of vaporization), rather it stems from the different values used for the hotplate area. Further, R11
262	changed their theoretical $f_2$ to an actual conversion factor that was "lower because of the
263	imperfect heat transfer from the precipitation to the hot plate (losses to the air, e.g.)." We do not
264	find justification for this in R11, nor do we agree with R11's assignment of $A_h = 0.008844 \text{ m}^2$
265	assuming they were recommending that value for the hotplate sold by YES. Recently, Boudala et
266	al. (2014) addressed the second of these two points, making it clear that $A_h = 0.01327 \text{ m}^2$ is
267	appropriate for the hotplate sold by YES.
268	In light of the above, the ratio of our $f_2$ (Eq. 9a – b with $T = T_h = 0$ °C), divided by the
269	actual conversion factor in R11, is 0.86 for snow and 0.89 for rain. Since a derived precipitation
270	rate is proportional to $f_2$ (e.g., Eq. 2), we expect the ratio of a precipitation rate from the new
271	algorithm (assuming $T = T_h = 0$ °C), divided by a synchronous hotplate-derived precipitation
272	rate, to be between 0.86 and 0.89. Our expectation hinges on the assumption that the YES
273	algorithm has incorporated R11's surface area and R11's distinction between theoretical and
274	actual conversion factors.
275	We calculate $f_2$ in the new algorithm two ways: 1) In a comparison made to a hotplate-
276	derived accumulation, our $f_2$ is set to $2.66 \times 10^{-8}$ m J <sup>-1</sup> (snow) and $3.01 \times 10^{-8}$ m J <sup>-1</sup> (rain). These
277	values were obtained from Eq. 9a – b with $T = T_h = 0$ °C and are displayed as a dotted line in Fig.
278	3a. 2) In comparisons made to either a NOAH-II accumulation or to a laboratory reference
279	precipitation rate, we evaluate $f_2$ using Eq. 9a – b with a $T_h$ from Table 3 and with the hotplate-
280	measured ambient $T$ (Table 1). In addition to the step change due to the difference between the





- 281 latent heats of sublimation and vaporization, our conversion factor has a weak temperature
- dependence (Fig. 3a, solid line). This is accounting for the warming discussed in section 2. Also,
- in Fig. 3a we display the actual conversion factor from R11 (dashed line). Our classification of
- 284 measurements into snow and rain is discussed in a later section.
- 285 **3.8 Snow Particle Catch Efficiency**

286 The hotplate's snow particle catch efficiency (E; section 2) is accounted for using a wind speed-dependent function (R11). The function accounts for the fact that snow particles landing 287 288 on the hotplate can bounce, and be carried away by the wind, or be sheared off by the wind after they land. This conceptual description of catch on the hotplate is different from that used to 289 describe catch by weighing gauges where undercatch results because a subset of snow particles 290 291 are carried over the gauge by a vertically-accelerated flow (Nespor and Servuk, 1999; Thériault et al., 2012). Both R11 and G98 derive catch efficiencies as the ratio of two paired values of 292 liquid-equivalent accumulation, one obtained from the gauge of interest and the other obtained 293 from a second gauge operated inside a Double Fence Intercomparison Reference Shield (DFIR). 294 The snow particle catch efficiency functions applied here are both gauge- and location-295 dependent. For the UW hotplate (at GLE and OWL), and the NCAR hotplate (at BTL), we apply 296 the function recommended by YES (YES 2012, personal communication; hereafter Y12). Wind 297 speeds used in the efficiency calculation are the hotplate-derived U. In addition, the hotplate 298 299 catch efficiency function described by R11 (their Equation 6) was also applied. This is based on 300 the hotplate U adjusted to the 10-m level with a roughness length  $z_0 = 0.3$  m (G98, their Equation 301 4.3.1) and was only used in analysis of measurements made at OWL. The  $z_0$  we picked 302 corresponds to a surface with "Many trees, hedges, few buildings" (Panofsky and Dutton, 1984; 303 their Figure 6.2). This assignment is consistent with the presence of shrubs and trees (Steenburgh





304	et al., 2014), and a two-story barn, at the OWL site. The barn was located at the eastern edge of a
305	fallow field, 80 m west of the gauges at OWL. For the NOAH-II gauge, we applied a function
306	developed for an 8-inch (diameter) Alter-shielded gauge (G98; their Equation 4.7.1). Wind
307	speeds used in that calculation are from the hotplate (at GLE and BTL) or from an anemometer
308	(at OWL) (Campbell et al., 2016). Of course, we are assuming that the function from G98
309	mimics undercatch by our 12-inch (diameter) Alter-shielded NOAH-II gauge.
310	In Fig. 3b, we present the three catch efficiency functions (R11 with U adjusted to 10 m,
311	Y12, and G98). In this graph, the wind speed applied in the R11 function is the value plotted on
312	the abscissa multiplied by 2.9. This adjustment corresponds to the lowest installation of the
313	hotplate at OWL and decreases to 2.0 for measurements made after 20131217 <sup>1</sup> . In our
314	calculation of the R11 catch efficiency functions, the snow depth for the interval of interest
315	(20131211 to 20140129) was set equal to the average (0.7 m) derived using an ultrasonic snow
316	depth instrument operated at OWL (Campbell et al., 2016). This average, and the AGL altitudes
317	of the hotplate installation (Table 2), were used to derive the two wind-speed adjustment factors
318	(2.9 and 2.0). The basis for this calculation is G98's gauge-height correction formula (their
319	Equation 4.3.1).
320	Since the anemometer at OWL was operated at nearly the same height as the top of the
321	NOAH-II gauge (Steenburgh et al., 2014), and the G98 catch efficiency formula (their Equation
322	4.7.1) assumes speeds are measured at the height of the gauge opening, a vertical adjustment of
323	the wind speed was not factored into the G98 catch efficiencies.

<sup>&</sup>lt;sup>1</sup> Altitudes of the two hotplate installations are provided in Table 2.





325

# 326 4 – Testing and Calibration Results

## 327 4.1 – Warm-Cold Tests

328 Results from the warm-cold tests are described here. The derived  $T_h/\gamma$  pairs (section 3.5)

are in Table 3. The  $T_h$  values are 42.2 °C for the NCAR hotplate deployed at BTL (NCAR/BTL),

52.2 °C the UW hotplate deployed at (UW/GLE), and 65.5 °C for the UW gauge deployed at

331 OWL (UW/OWL). The first two  $T_h$ s differ from those presented in Wettlaufer (2013) where, for

the NCAR hotplate, he reported agreement with the nominal plate temperature (75 °C; R11) and

for the UW hotplate (GLE) he reported a larger temperature ( $T_h = 109 \text{ °C}$ ). The  $T_h/\gamma$  pair

reported in Table 3, for the UW/OWL study, was evaluated after Wettlaufer (2013) reported his

335 warm-cold test results.

In our analysis of the warm-cold measurements we only used data acquired in the hangar;

337 Wettlaufer (2013) analyzed both hangar and lab data. For us the warm-cold temperature pairings

338 are 5.4/-4.3 °C (NCAR/BTL), 7.0/-1.1 °C (UW/GLE), and 29.5/10.4 °C (UW/OWL). Compared

to Wettlaufer (2013), our  $T_w$ s are 15 °C colder (NCAR/BTL and UW/OWL experiments only).

Using our  $T_h/\gamma$  pairs (Table 3) and the first two  $T_w$ s (i.e., for NCAR/BTL and UW/GLE), we

evaluated the term in Eq. 7a representing natural-convective transfer  $(D_h \cdot K_x \cdot (T_h - T_w) \cdot \gamma)$  and

342 compared to values derived using  $T_h/\gamma$  pairs in Wettlaufer (2013; his Table 2). In the NCAR/BTL

343 comparison  $T_w$  was set at 5.4 °C and in the UW/GLE comparison  $T_w$  was set at 7.0 °C. Our

natural-convective term agrees within  $\pm 0.1$  W of those derived by Wettlaufer (2013). Also in

good agreement is the product of  $T_h$  and  $\gamma$ . Relative to Wettlaufer (2013), our  $T_h \times \gamma$  product is 6

346 % larger (NCAR/BTL), and 7 % larger (UW/GLE). We expect that our  $T_h/\gamma$  pairs (Table 3),





347 when applied in Eq. 3, will produce a reasonable estimate of the precipitation rate. We test that

348 expectation in the next section.

Error limits on  $T_h$  and  $\gamma$ , in Table 3, were derived by perturbing  $Q_{top,w}$  (i.e., the value acquired in the warm test) by  $\pm 0.5$  W and repeating the analysis (Eq. 7a - b). Our estimate of the  $Q_{top,w}$  error ( $\pm 0.5$  W) came from a comparison of values acquired before and after power to the hotplate was stopped and restarted. These tests were conducted in the hangar and the 10 min warm up recommended by the manufacturer was adhered to (YES, 2011).

## 354 4.2 - Drip Tests

355 This section compares two time sequences of precipitation rate: one calculated with the 356 new algorithm, the other is the hotplate-derived value (Table 1). The basis for the comparison is 357 measurements of artificially-produced liquid precipitation made in the hangar. We applied water 358 drops to the NCAR and UW hotplates using a volumetric water pump (Ismatec Inc.; Model 7618). Each of these tests has a drip period (4 min) and a nondrip period (5 min). Drops (4 mm 359 volume-equivalent diameter) were added uniformly to the top plate at a constant volumetric rate. 360 Assuming all of the pumped water is delivered to the hotplate, the pump rate is proportional to a 361 precipitation rate. We see no reason to question this assumption. Hence, we convert the pump 362 rate to a reference precipitation rate  $(P_{REF})$  and apply the  $P_{REF}$  in subsequent analyses<sup>2</sup>. These 363 drip tests were conducted at T > 4 °C. 364 365 Because the drip tests were conducted with the hotplate operating as in Fig. 2, and

- unventilated, the recorded data were analyzed with  $T_s = T$ , in Eq. 6 (section 3.5), and with the
- sensible power output formulated as  $D_h \cdot K_x \cdot (T_h T) \cdot \gamma$  (Appendix and Table 3). Also, because all

<sup>&</sup>lt;sup>2</sup> The value of the multiplier that converts the volumetric pump rate ( $cm^3 min^{-1}$ ) to precipitation rate ( $mm hr^{-1}$ ) is 4.51.





369 constraints, precipitation rates were derived by inputting measurements ( $Q_{top}$ , <i>T</i> , <i>U</i> , and 370 a calculated variable ( $IR_d$ ; section 3.4) into Eq. 3 and solving for a precipitation rate seq 371 ( $P(t)$ ). We symbolize this $P(t)$ as $P_{UW}$ and refer to calculations leading to that sequence 372 UW algorithm. Also, we refer to sequences obtained from the UHP file (Table 1) as $P_{TH}$	SW) and uence
a calculated variable ( $IR_d$ ; section 3.4) into Eq. 3 and solving for a precipitation rate seq ( $P(t)$ ). We symbolize this $P(t)$ as $P_{UW}$ and refer to calculations leading to that sequence UW algorithm. Also, we refer to sequences obtained from the UHP file (Table 1) as $P_{TH}$	uence
371 ( $P(t)$ ). We symbolize this $P(t)$ as $P_{UW}$ and refer to calculations leading to that sequence 372 UW algorithm. Also, we refer to sequences obtained from the UHP file (Table 1) as $P_{YH}$	
372 UW algorithm. Also, we refer to sequences obtained from the UHP file (Table 1) as $P_{YH}$	as the
	zs and
373 refer to that calculation as the YES algorithm.	
374 We now compare values of $P_{UW}$ to synchronous values of $P_{YES}$ . Typically, these	rates
exhibit a maximum ~ 3 min after the nondrip-to-drip transition (Fig. 4). We interpret the	ese
376 maxima as overestimates, possibly due to a violation of the steady-state assumption. Al	SO
evident, particularly in the $P_{UW}$ sequence, is a minimum. This occurs during the time the	9
378 instrument is relaxing to its rest state; i.e. ~ 2 min after a drip-to-nondrip transition. The	figure
also demonstrates that thresholding is applied to the $P_{YES}$ sequence, i.e. the YES algorith	ım
380 thresholds the output to 0 mm hr <sup>-1</sup> if values decrease to $< 0$ mm hr <sup>-1</sup> . This is evident at ~	· 16:11
381 UTC and at three other times in the $P_{YES}$ sequence.	
382 Two 1-min averaging intervals are shown in Fig. 4. We set the end of these at th	e drip-to-
nondrip transitions. Fig. 5 is a compilation of the two tests already discussed (Fig. 4) pla	us four
additional $P_{REF}$ vs $\langle P_{UW} \rangle$ comparisons and four additional $P_{REF}$ vs $\langle P_{YES} \rangle$ comparisons	
385 We now use linear least-squares regression analysis, and a regression equation of	of form y
$a = a \cdot x$ , to derive the ratio of two precipitation rates. In Fig. 5 it is apparent that the regres	ssion
slope (ratio), derived for the $P_{REF}$ vs $\langle P_{UW} \rangle$ comparison, does not differ from one by me	ore than $\pm$
388 1 standard deviation. Ratios for the two hotplates (UW and NCAR) and for three drip te	sts are
summarized in Table 4. In the third column ( $P_{REF}$ vs $\langle P_{UW} \rangle$ ), we see that none of the ra	tios differ
from one by more than $\pm 1$ standard deviation. Different from Fig. 5 and Table 4, we also	<b>SO</b>





- evaluated intercepts of regressions that were not forced through the origin; none of these
- 392 intercepts differ significantly from zero (results not shown). From the statistical comparisons in
- Table 4, we conclude the  $T_h/\gamma$  pairs (Table 3) applied in the UW algorithm (Eq. 3) produce a
- 394 precipitation rate consistent with the reference.
- 395 Values of the reference rate and the hotplate-derived rate  $(\langle P_{YES} \rangle)$  are compared as ratios
- in Fig. 5 and in the fourth column of Table 4. These ratios are seen to deviate systematically
- from unity and in the direction discussed in section 3.7. In the unforced regressions (not shown)
- the intercepts are negative, but only one of these differed significantly from zero (NCAR/BTL;
- intercept =  $-0.3 \pm 0.1$ ). Negative intercepts are expected because  $P_{YES}$  is positively offset, by ~
- 400  $0.2 \text{ mm hr}^{-1}$ , during most of the nondrip periods (e.g., 16:21 UTC in Fig. 4).

#### 401 5 - Field Measurements

- 402 This section is organized as follows: Section 5.1 presents field measurements of ambient
- 403 temperature and ambient ice-bulb temperature. We use this information to classify 27
- 404 precipitation events as snowfall or rainfall. Section 5.2 presents the Nu-Re relationship we use to
- 405 account for the sensible power term in Eq. 3. Section 5.3 describes how we derive a precipitation
- 406 rate for a hotplate based on measurements made in the field. Section 5.4 compares time-
- 407 integrated precipitation rates (accumulations) derived using the two algorithms. In section 5.4,
- 408 we also compare hotplate accumulations to values from the NOAH-II.

## 409 5.1 – Field-measured Temperatures and Ice-bulb Temperatures

- 410 The 27 precipitation events are summarized in Table 5. Measurements were made during
- 411 2012, at the two Southeast Wyoming field sites (BTL and GLE), and during 2013 and 2014 at
- 412 the Western New York site (OWL). Table 5 and Fig. 6 have event-averaged ambient
- 413 temperatures ( $\langle T \rangle$ ) and the event-averaged ambient ice-bulb temperatures ( $\langle T_{IB} \rangle$ ; section 3.1).





- 414 Twenty-three of the events have  $\langle T \rangle \leq -3.3$  °C and upper-limit temperature ( $\langle T \rangle$  plus two
- 415 standard deviations) no warmer than -2.3 °C. We classified these as snowfall. In addition, we
- 416 classified four events as rainfall. These had  $\langle T_{IB} \rangle \ge +2.9$  °C and lower-limit temperature ( $\langle T \rangle$
- 417 minus two standard deviations) no colder than +2 °C.
- 418 5.2 Nusselt-Reynolds Relationship

419 Fig. 7b shows a plot of the *Nu-Re* fit function with the data used to constrain the function.

420 This result is for the UW hotplate operating at the GLE site. Fit coefficients ( $\alpha$ ,  $\beta$ , and  $\gamma$ ) are

421 reported in Table 6 for each field site. Hansen and Webb (1992) reported  $\alpha = 0.09$  and a  $\beta$ 

422 between 0.69 and 0.72 for a surface similar to the hotplate (circular with three concentric rings);

423 however, their flow direction was perpendicular to the plate surface. The values of  $\alpha$  and  $\beta$  we

report may differ from those in Hansen and Webb (1992) because the flow is principally parallel

425 to the plate surface at our field sites. There are two other differences relative to Hansen and

426 Webb (1992): 1) Our geometrically-averaged *Nu* (~ 360) is about a factor of five larger, and 2)

427 our *Re* extends smaller by a factor of two and larger by a factor of three. Finally, we note that

428 compared to Fig. 7b there is an order of magnitude narrower *Re* range in the NCAR/BTL and

429 UW/OWL *Nu-Re* plots (not shown).

Fig. 7a is a companion to Fig. 7b showing the  $\gamma$  based on the warm-cold test. The error limit on this datum is explained in section 4.1. Since *Nu* is dependent on the *T<sub>h</sub>* derived in the warm-cold test (section 3.5), we expect the *Nu-Re* function to converge to the warm-cold  $\gamma$  in the limit of small *Re*. In our assessment of convergence, we evaluated the limiting *Nu* at the *Re* corresponding to the minimum *U* reported in the hotplate data output (0.1 m s<sup>-1</sup>). This minimum *U* establishes the left end of the function in Fig. 7b. Convergence of the *Nu-Re* relationship to

436 within the error limit on the warm-cold  $\gamma$ , at the former's left-most limit, is evident in Fig. 7a – b.





437 Convergence is also evident in the NCAR/BTL and UW/OWL plots analogous to Fig. 7a – b

438 (not shown) and this in spite of narrower *Re* range in those datasets.

## 439 **5.3 - Precipitation Rate from Field Measurements**

Fig. 8 shows energy-budget terms (Eq. 3) for one of the four rainfall events in our dataset
(OWL-15 in Fig. 6). The three output terms (sensible, latent, and upwelling longwave), and three

442 input terms (top plate power, downwelling longwave, and shortwave) are shown in Fig. 8a - b. In

443 this section we begin with the sequence of latent power output (i.e., the sequence labeled  $P \cdot E/f_2$ 

444 in Fig. 8a) and describe how we calculate the sequence of rainfall rate. We also contrast that

445 calculation with steps followed in the case of snowfall.

The first step in the calculation is conversion of the latent term (Fig. 8a) to a provisional

447 precipitation rate; this is done by multiplying each element of the latent term by the

448 corresponding element of  $f_2$  (Eq. 9b). This operation is referred to as element-by-element vector

449 multiplication. Thresholding is applied next. Both a 300-s running average of the provisional rate

450 and a 10-s running average of the provisional rate are computed. If the 300-s average exceeds

451  $0.25 \text{ mm hr}^{-1}$ , and the 10-s average exceeds 0 mm hr<sup>-1</sup>, the rate is stored as the 10-s average;

452 otherwise the rate is stored as 0 mm hr<sup>-1</sup>. We refer to the resultant as  $P_{UW}$ , but we note that in

453 section 4.2 the  $P_{UW}$  sequences were unthresholded. Both the thresholded and unthresholded

454 sequences are presented in Fig. 8c - d. The thresholded  $P_{UW}$  is identical to the unthresholded  $P_{UW}$ 

455 where the 300-s average exceeds  $0.25 \text{ mm hr}^{-1}$  and the 10-s average exceeds  $0 \text{ mm hr}^{-1}$ .

456 In the case of snowfall, the  $f_2$  is calculated using Eq. 9a and applied as discussed in the

457 previous paragraph. Finally, the precipitation rate is derived as the resultant of element-by-

458 element vector multiplication of the thresholded  $P_{UW}$  and the reciprocal of the snow particle

459 catch efficiency (section 3.8).





## 460 5.4 – Comparisons of Liquid-equivalent Accumulation

461	Here we use linear least-squares regression analysis, with a regression equation of form y
462	$= a \cdot x$ , to derive the ratio of two measures of liquid-equivalent accumulation for snow. In Fig. 9,
463	these measures are the accumulations derived using the UW and YES algorithms. In the these
464	algorithms the particle catch efficiency function is the one described in Y12 and $f_2$ is $2.66 \times 10^{-8}$
465	m J <sup>-1</sup> (section 3.7). The data points correspond to measurements made at GLE (UW hotplate), at
466	BTL (NCAR hotplate), and at OWL (UW hotplate). We note that 19 of 23 y-axis values are from
467	the same instrument (UW hotplate) and are derived using the same calibration (UW/OWL)
468	applied to produce the result shown in the third row of Table 4. Statistical consistency between
469	the ratio in Fig. 9 (0.79 $\pm$ 0.05) and the ratio in the third row of Table 4 (i.e., 0.79 $\pm$ 0.03 for the
470	$P_{REF}$ vs $\langle P_{YES} \rangle$ ratio) suggests a systematic error in the YES-derived precipitation rates and
471	accumulations. This assertion is reinforced by the three NCAR hotplate points straddling the
472	best-fit line, in Fig. 9, and by the ratio reported in Table 4 for the NCAR hotplate (i.e., 0.81 $\pm$
473	0.03 for the $P_{REF}$ on $\langle P_{YES} \rangle$ ratio). However, we cannot exclude the possibility that bias in our
474	field-based calibration coefficients ( $\alpha$ , $\beta$ , and $\gamma$ ; Table 6) is the reason for a UW/YES ratio
475	significantly smaller than unity in Fig. 9.
476	As was discussed in section 4.2, and demonstrated in Fig. 4, during the indoor nondrip
477	periods the $P_{YES}$ sequence is positively offset. A plausible reason for this, and for the ratios < 1
478	reported in the previous paragraph, is disregard for longwave forcing in the YES algorithm.

479 Since we do not have access to the YES algorithm, we estimated the longwave radiative effect

480 by setting the downwelling and upwelling longwave terms to zero in Eq. 3. After doing this, a

481 larger UW/YES ratio was obtained in a plot analogous to Fig. 9 ( $0.83 \pm 0.04$ ). From this modest

482 increase of the UW/YES ratio, we conclude that offset due to longwave forcing *cannot* explain





483	UW/YES ratios significantly smaller than unity. An even smaller-magnitude perturbation of the
484	UW/YES ratios was obtained when zeroing the shortwave term in Eq. 3 (results not shown).
485	Further evidence for systematic error in the YES values comes from Fig. 10. With the
486	exception that these data are for rain observed at OWL (section 5.1) the comparison in Fig. 10 is
487	similar to Fig. 9. Although the number of points is small, Fig. 10 establishes that our finding of a
488	UW/YES ratio significantly smaller than unity is true for both rainfall and snowfall. In addition,
489	Fig. 11 strengthens this conclusion by showing agreement between values of UW algorithm
490	accumulation and the NOAH-II accumulation when both gauges are detecting rain.
491	An additional assessment of snowfall at OWL is presented in Fig. $12a - c$ . In these graphs
492	NOAH-II measurements are plotted on the abscissa and different interpretations of the UW
493	hotplate measurements are plotted on the ordinate. For both devices, we plot the ratio of a liquid-
494	equivalent accumulation divided by an event-averaged particle catch efficiency, and we note that
495	the numerator of this ratio is an accumulation that was not corrected for inefficient catch <sup>3</sup> and
496	that values contributing to these ratios are in Table 5. Table 5 demonstrates two features of the
497	OWL snow data set: 1) The event-averaged catch efficiency based on Y12 (< E Y12>) is
498	consistently larger than the event-averaged efficiency based on R11 ( $\langle ER11 \rangle$ ), and 2) the
499	event-averaged efficiency $\langle E R I I \rangle$ is comparable to $\langle E Y I 2 A n \rangle$ , where the latter is the event-
500	averaged efficiency derived with the anemometer $U$ and the Y12 catch efficiency function. These
501	features are consistent with the altitude adjustment in R11, which increases the wind speed
502	(section 3.8), and thus decreases $\langle E R11 \rangle$ relative to $\langle E Y12 \rangle$ . They are also consistent with a
503	low bias in the hotplate-derived $U$ . The latter is supported by a comparison of the hotplate $U$ vs

<sup>&</sup>lt;sup>3</sup> This comparison was also made using accumulations corrected with a time-dependent catch efficiency (section 5.3), but we found that the fit-line slopes differed by less than  $\pm$  5 % from those in Fig. 12.





anemometer U where the fit-line slope is  $0.55 \pm 0.05$  for the 19 snow events at OWL (results not

505 shown).

506 Consistent with the ranking of event-averaged Es (Table 5), Fig. 12a shows that the 507 hotplate values, derived with the hotplate U and the Y12 catch efficiency function, are statistically smaller than the NOAH-II-derived values. We also see that the 15% statistical 508 underestimate in the hotplate (Fig. 12a) reverses to a slight overestimate when using the R11 509 510 catch efficiency function (Fig. 12b) and when using the anemometer U with the Y12 function 511 (Fig. 12c). Unfortunately, these results do not allow us to specify relative contributions to the 512 15% statistical underestimate (Fig. 12a) coming from the fact that the Y12 function does not use 513 a height-adjusted U or from the suspected hotplate underestimate of U. Further studies focused on development of a hotplate catch efficiency function dependent on the local wind speed, as 514 opposed to the wind speed at 10 m (R11), and investigation of the hotplate's determination of 515 516 wind speed, are needed to resolve this issue.

## 517 6 - Conclusions

Measurements made with two YES hotplates were used to derive precipitation rates and 518 accumulations for 27 snowfall and rainfall events. The basis for this is an energy budget equation 519 similar to that in King et al. (1978). We changed that energy budget (Eq. 1) by including terms 520 that describe longwave and shortwave radiant energy transfer (Eq. 3). To the best of our 521 522 knowledge, this is the first time that radiative terms have been incorporated into a hotplate data 523 analysis algorithm and reported in the scientific literature. In this paper, we have used computational methods different from those in R11, and we 524 derived and applied different calibration coefficients. In spite of these changes we report 525

526 precipitation rates and accumulations that strongly correlate with the output of two YES





527	hotplates. However, a systematic difference is evident in comparisons of the UW and YES
528	algorithms. We surmise that the difference comes from the following: 1) R11's assignment of $A_h$
529	$(0.00884 \text{ m}^2 \text{ vs } 0.01327 \text{ m}^2 \text{ in the UW algorithm}), 2)$ R11's distinction between a theoretical and
530	an actual energy conversion factor, and 3) the incorporation of #1 and #2 into the YES algorithm.
531	Clearly, R11's $A_h$ is not justified for hotplates sold by YES (Boudala et al., 2014; YES 2017,
532	personal communication). R11's distinction between conversion factors is more problematic.
533	That distinction can be interpreted two ways: either 1) The distinction accounts for
534	environmental thermal energy input that assists the conversion of precipitation mass to vapor, or
535	2) the distinction accounts for the loss of snow particles from the top surface of the hotplate due
536	to removal by wind. Because early in the warming process a precipitation element attains a
537	temperature larger than that of the air, we assert that the first of these phenomena is unlikely to
538	contribute significantly to the energy budget. The second may be significant, but it is our opinion
539	that removal of precipitation mass by wind is best accounted with a catch efficiency, not with a
540	distinction between conversion factors. Lastly, accounting for either of these phenomena,
541	independent of an adjustment of the catch efficiency, should be accomplished with an increase of
542	an actual conversion factor relative to the theoretical value, not with the decrease proposed by
543	R11.
544	





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555	Appendix – I	Nomenclature
556	$A_h$	Area of YES hotplate = $0.01327 \text{ m}^2$
557	С	Liquid H <sub>2</sub> O specific heat capacity = $4218 \text{ J kg}^{-1} \text{ K}^{-1}$ (assumed independent of
558		temperature; Iribarne and Godson, 1981; their Table IV-5)
559	$C_i$	Solid H <sub>2</sub> O specific heat capacity = $2106 \text{ J kg}^{-1} \text{ K}^{-1}$ (assumed independent of
560		temperature; Iribarne and Godson, 1981; their Table IV-5)
561	$D_h$	Diameter of YES hotplate = $0.130 \text{ m}$
562	Ε	Snow particle catch efficiency (section 3.8)
563	$f_1$	Wind speed-dependent property in Eq. 2 [W]
564	$f_2$	Electrical-to-precipitation conversion factor [m J <sup>-1</sup> ]
565	IR	Component of longwave flux [W m <sup>-2</sup> ]
566	$L_f(T_o)$	Latent heat of fusion evaluated at the thermodynamic reference temperature =
567		0.3337x10 <sup>6</sup> J kg <sup>-1</sup> (Iribarne and Godson, 1981; their Table IV-5)
568	$L_v(T_h)$	Latent heat of vaporization at $T_h$ (Iribarne and Godson, 1981; their Equation
569		4.103) [J kg <sup>-1</sup> ]
570	M <sub>IR</sub>	Measured net longwave flux (section 3.4) [W m <sup>-2</sup> ]
571	Nu	Nusselt number
572	Р	Liquid-equivalent precipitation rate [mm hr-1 or m3 m-2 s-1]
573	$P_{Ref}$	Reference precipitation rate (section 4.2) [mm hr <sup>-1</sup> or m <sup>3</sup> m <sup>-2</sup> s <sup>-1</sup> ]
574	$P_{UW}$	Precipitation rate derived with UW algorithm (section 5.3) $[mm hr^{-1} \text{ or } m^3 m^{-2} s^{-1}]$
575	$P_{YES}$	Precipitation rate derived with YES algorithm (section 4.2) [mm hr <sup>-1</sup> or $m^3 m^{-2} s^{-1}$ ]
576	$Q_{bot}$	Bottom plate power [W]
577	$Q_{top}$	Top plate power [W]
578	$R_d$	Dry air specific gas constant = $287 \text{ J kg}^{-1} \text{ K}^{-1}$
579	Re	Reynolds number
580	$R_h$	Hotplate Reflectance = $0.63$ (section 2.2)
581	SW	Measured shortwave flux (section 2.2) [W m <sup>-2</sup> ]
582	Т	Ambient temperature [°C or K]
583	$T_h$	Hotplate surface temperature (section 3.5) [°C or K]





584	$T_o$	Thermodynamic reference temperature = $0.0 \ ^{\circ}C$
585	$T_s$	Temperature of painted-steel sheeting [°C or K]
586	U	Wind speed [m s <sup>-1</sup> ]

# 587 Greek Symbols

588	α	Fitted Nu-Re Coefficient (section 3.6)
589	β	Fitted Nu-Re Coefficient (section 3.6)
590	$\mathcal{E}_h$	Hotplate emissivity = $0.14$ (section 2.2)
591	$\mathcal{E}_{S}$	Emissivity of painted-steel sheeting = $0.84$ (section 3.3)
592	γ	Coefficient derived in warm-cold tests (section 3.5) or a coefficient in the Nu-Re
593		relationship (section 5.2)
594	ρ	Liquid H <sub>2</sub> O density = $1000 \text{ kg m}^{-3}$ (assumed independent of temperature)
595	$\sigma$	Stefan-Boltzmann constant = $5.67 \times 10^{-8}$ W m <sup>-2</sup> K <sup>-4</sup>

# 596 Subscripts

597	С	Indoor cold setting
598	d	Downwelling
599	IB	Ice-bulb
600	S	Painted-steel sheeting
601	и	Upwelling
602	w	Indoor warm setting
603	x	Property of air film adjacent to the hotplate surface: $p_x =$ standard-atmosphere
604		pressure at the altitude of the measurement. The following three film properties
605		are held constant in calculation of the Reynolds number (section 3.6) and in
606		calculation of the sensible power output due to molecular conduction (section
607		3.6): 1) temperature ( $T_x = 303.15$ K), 2) dynamic viscosity ( $\mu_x = 1.862 \times 10^{-5}$ kg m <sup>-</sup>
608		<sup>1</sup> s <sup>-1</sup> ; Rogers and Yau (1989; their Table 7.1)), and 3) thermal conductivity ( $K_x =$
609		$2.63 \times 10^{-2}$ J m <sup>-1</sup> s <sup>-1</sup> K <sup>-1</sup> ; Rogers and Yau (1989; their Table 7.1)).

# 610 **Operator**

611 <y> '</y>	Time average	of property	y
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Figure 3 – a) Electrical-to-precipitation conversion factors vs ambient temperature assuming snow at T < 0 °C and rain at T > 0 °C. See text for details. b) Snow particle catch efficiency vs wind speed using the R11, Y12, and G98 formulations discussed in the text.







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Figure 4 – Precipitation rates, derived using the UW and YES algorithms, plotted vs time. The figure shows two drip periods (starting at the nondrip-to-drip transitions), both with the reference set at 3.1 mm hr<sup>-1</sup>, and nondrip periods (starting at the drip-to-nondrip transitions). Dashed vertical lines indicate the transitions and 1-min precipitation averaging intervals. Measurements are from the UW hotplate operating indoors on 20120229. The UW/GLE calibration constants (Table 3) and an  $f_2$  derived with the second of two methods (section 3.7)

were applied in the UW algorithm.

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Figure 5 - Reference precipitation rate vs time-averaged  $P_{UW}$  and  $P_{YES}$ . Measurements are from the UW hotplate operating indoors on 20120229. The UW/GLE calibration constants (Table 3) and an  $f_2$  derived with the second of two methods (section 3.7) were applied in the UW algorithm. Regression lines were forced through the origin and *x* deviations (horizontal departures of data from regression line) were used as the basis for the least squares criterion of best fit (Young, 1962). Standard deviations on the fitted ratios (confidence intervals) were derived using Student's t-distribution at the 95% level (Havilcek and Crain, 1988).







827 in Table 5. Error bars are  $\pm 2$  standard deviations. The dashed horizontal line is drawn at +4 °C.

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848 hotplate at the GLE site. For clarity, only every fortieth *Nu-Re* data pair is plotted. The minimum

849 *Re* plotted (data and fit function) corresponds to the minimum *U* reported in the UHP file (0.1 m  $s^{-1}$ ). The measurement interval is 20120402 04:00 UTC to 20120402 09:00 UTC at the GLE site.

The UW/GLE  $T_h$  (Table 3) was applied in the data analysis.

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Figure 9 – Snow accumulations derived using the UW algorithm vs snow accumulations derived using the YES algorithm. Both the Y12 catch efficiency function and an  $f_2$  derived with the first of two methods discussed in section 3.7 were applied in the UW algorithm. The regression line was forced through the origin and y deviations (vertical departures of data from regression line) were used as the basis for the least squares criterion of best fit (Young, 1962). The standard deviation on the fitted ratio (confidence interval) was derived using Student's tdistribution at the 95% level (Havilcek and Crain, 1988).







906Figure 10 - Rain accumulations derived using the UW algorithm vs rain accumulations907derived using the YES algorithm. An  $f_2$  derived with the first of two methods discussed in section9083.7 was applied in the UW algorithm. Regression lines were forced through the origin and y909deviations (vertical departures of data from regression line) were used as the basis for the least910squares criterion of best fit (Young, 1962). The standard deviations on the fitted ratios911(confidence intervals) were derived using Student's t-distribution at the 95% level (Havilcek and912Crain, 1988).







Figure 11 –Rain accumulations derived using the UW algorithm vs rain accumulations from the NOAH-II gauge. An  $f_2$  derived with the second of two methods discussed in section 3.7 was applied in the UW algorithm. Regression lines were forced through the origin and y deviations (vertical departures of data from regression line) were used as the basis for the least squares criterion of best fit (Young, 1962). The standard deviations on the fitted ratios (confidence intervals) were derived using Student's t-distribution at the 95% level (Havilcek and Crain, 1988).

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- 954 Figure 12 UW hotplate and NOAH-II measurements of snow (liquid-equivalent
- 955 accumulations, not corrected for inefficient catch, divided by an event-averaged snow particle
- 956 catch efficiency) at OWL. An  $f_2$  derived with the second of two methods discussed in section 3.7
- 957 was applied in the UW algorithm. Regression lines were forced through the origin and *y*
- 958 deviations (vertical departures of data from regression line) were used as the basis for the least
- squares criterion of best fit (Young, 1962). The standard deviations on the fitted ratios
- 960 (confidence intervals) were derived using Student's t-distribution at the 95% level (Havilcek and
- 961 Crain, 1988).





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963	Table 1 -	- Hotplate	Data	Files
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965	Recorded Variable <sup>a</sup> , dimension	File	File	Symbol
	interested variable , anteriston	UHP	SHP	2 jiiio or
966	Unix Time, s	√	✓	
0.07	Precip. Rate, mm hr <sup>-1</sup>	$\checkmark$		$P_{YES}$
967	Accumulated Precip., mm	✓		
068	Ambient Temp., °C	~	~	Т
908	Enclosure Temp., °C	~	~	
969	Wind Speed, m s <sup>-1</sup>	~		U
303	Shortwave Radiation, W m <sup>-2</sup>	~	~	SW
970	Net Longwave Radiation, W m <sup>-2</sup>	~	~	$M_{IR}$
570	Barometric Pressure, hPa	~	~	$p^{b}$
971	RH Sensor Temp., °C	~	~	
	RH, %	$\checkmark$	$\checkmark$	RH
972	Top Plate Voltage, V		$\checkmark$	
-	Bottom Plate Voltage, V		$\checkmark$	
973	Top Plate Current, A		$\checkmark$	
	Bottom Plate Current, A		$\checkmark$	
974	Top Plate Resistance, $\Omega$		~	
	Bottom Plate Resistance, $\Omega$		$\checkmark$	
975	Top Plate Power, W		$\checkmark$	$Q_{top}$
	Bottom Plate Power, W		$\checkmark$	$Q_{bot}$
976	Radiation Sensors' Temp., °C		~	

<sup>a</sup> With the exception of Unix time, all recorded variables are 60-s running averages, sampled at 1 Hz
(YES, 2011)

979 <sup>b</sup> Although pressure is a recorded variable, the pressure used in the UW algorithm ( $p_x$ ; section 3.6 and

980 Appendix) is the standard-atmosphere pressure at the altitude of the measurement





## Table 2 – Field Sites, Site Location, Vegetation at the Site, Gauge Location, Number of Events, and Event Type

Site Abbreviation Site Reference Hotplate	Site Location	Height of Vegetation, m AGL	Gauge Location at Site	Precipitation Events
GLE Wettlaufer (2013) UW	SE Wyoming 106.240 °W 41.3665 °N 3190 m MSL	10 to 20 m	) to 20 m Hotplate: 27 m AGL on top deck of a meteorological tower <sup>a</sup> NOAH-II: 3 m AGL (clearing in conifer forest 80 m SE of tower)	
BTL Wettlaufer (2013) NCAR	SE Wyoming 106.975 °W 41.1558 °N 3010 m MSL	10 to 20 m	Clearing in conifer forest Hotplate: 3 m AGL NOAH-II: 3 m AGL	3 Snow
OWL Steenburgh et al. (2014) UW	NW New York 75.8771 °W 43.6245 °N 385 m MSL	2 to 5 m	Clearing in deciduous brush and deciduous trees Hotplate: 1.7 m AGL <sup>b</sup> and 2.5 m AGL <sup>c</sup> NOAH-II: 2.5 m AGL	4 Rain 19 Snow

<sup>a</sup> This is the Brooklyn Lake, Wyoming AmeriFlux Tower; AmeriFlux is a network of sites that measure energy and trace-gas fluxes.

<sup>b</sup> First 7 of 23 OWL precipitation events (Date < 20131217)

<sup>c</sup> Last 16 of 23 OWL precipitation events (Date > 20131217)





# Table 3 – Summary of Warm-cold Tests

Indoor Calibration (Warm-cold Tests)						
Year Hotplate/Field Site $T_{hp}$ , $\gamma$						
2012	NCAR/BTL	$42.2 \pm 7.4$ <sup>a</sup>	106. ± 15.1 <sup>a</sup>			
2012 - 2013	UW/GLE	$52.2 \pm 15.7$	$74.8 \pm 18.1$			
2013 - 2015	UW/OWL	$66.5 \pm 7.8$	$57.8 \pm 7.7$			

<sup>a</sup> Error limits derived by perturbing  $Q_{top,w}$  (i.e., the value acquired in the warm test) by  $\pm 0.5$  W and repeating the analysis based on Eq. 7a - b.





Table 4 – Summary of Drip Tests

Indoor Calibration (Drip Tests)						
Year	Hotplate/Field Site	$P_{REF}$ vs $<\!\!P_{UW}\!\!>$ ratio <sup>a</sup>	$P_{REF}$ vs $<\!\!P_{YES}\!\!>$ ratio <sup>a</sup>	# <sup>b</sup>		
2012	NCAR/BTL	$0.99\pm0.02$	$0.81\pm0.03$	6		
2012 - 2013	UW/GLE	$1.00\pm0.06$	$0.84\pm0.05$	6		
2013 - 2015	UW/OWL	$0.97\pm0.04$	$0.79\pm0.03$	6		

<sup>a</sup> Ratios were derived as the slope of a regression lines forced through the origin. The *x* deviations (horizontal departures of data from regression line) were used as the basis for the least squares criterion of best fit (Young, 1962). Standard deviations on the fitted ratios (confidence intervals) were derived using Student's t-distribution at the 95% level (Havilcek and Crain, 1988).

 $^{b}$  # = number of tests.

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	770	1-10							ſ				
Precinitation	Date	Time	Time	a a	<tm><sup>b</sup></tm>	<11> c	p <b>M</b> 11	VES d		c		NOAH-II d	
Event	YYYYMMDD UTC	HH:MM UTC	HH:MM UTC	ς Υ	°C ,	m s <sup>-1</sup> ,	, mm	, mm	<e y12=""><sup>€</sup></e>	<e an="" y12=""><sup>†</sup></e>	<e r11=""><sup>g</sup></e>	mm	<e g98=""><sup>h</sup></e>
GLE-01	20120414	0:00	13:00	-5.1	-5.6	1.8	5.6	6.0	0.84	NA <sup>i</sup>	i AAP j	7.3	0.89
BTL-01	20120116	11:00	1:00	-10.2	-10.5	2.0	3.8	7.5	0.80	NA	NAP	8.0	0.87
BTL-02	20120119	8:00	18:00	-5.4	-5.6	4.3	3.8	3.4	0.55	NA	NAP	1.0	0.60
BTL-03	20120120	7:00	18:00	-5.0	-5.2	2.4	6.3	7.1	0.75	NA	NAP	8.2	0.83
OWL-01	20131211	18:00	0:00	-6.5	-6.9	1.4	16.9	21.5	0.91	0.64	0.70	20.0	0.72
OWL-02	20131212	0:00	6:00	-10.9	-10.7	0.4	1.0	3.5	0.99	0.94	06.0	0.3	0.96
OWL-03	20131212	6:00	12:00	-22.5	-21.6	0.3	0.0	2.4	1.00	1.00	0.93	0.0	1.00
OWL-04	20131212	18:00	0:00	-9.8	-10.5	1.3	0.2	1.2	06.0	0.53	0.67	0.3	0.57
OWL-05	20131213	6:00	12:00	-8.4	-9.0	0.9	13.9	18.5	0.96	0.73	0.80	12.1	0.81
0WL-06	20131215	19:45	0:00	-6.3	-6.9	0.9	1.3	2.9	0.94	0.85	0.78	0.5	0.88
OWL-07	20131216	0:00	6:00	-6.4	-7.3	0.3	20.9	27.8	1.00	1.00	0.93	15.2	1.00
OWL-08	20131218	18:00	0:00	-3.3	-3.7	1.5	10.4	13.2	0.88	0.82	0.77	12.9	0.87
OWL-09	20140106	18:00	0:00	-6.2	-6.5	3.9	4.5	4.6	0.58	0.42	0.39	2.5	0.35
OWL-10	20140107	0:00	6:00	-14.4	-14.7	3.8	0.4	0.3	0.60	0.44	0.42	0.0	0.38
OWL-11	20140107	6:00	12:00	-17.3	-17.4	3.0	1.9	3.3	0.67	0.49	0.52	4.0	0.49
OWL-12	20140107	18:00	0:00	-17.5	-17.9	3.9	0.0	0.0	0.58	0.42	0.39	0.3	0.35
OWL-13	20140111	15:45	21:00	7.2	6.6	1.1	11.9	14.1	1.00	1.00	1.00	13.2	1.00
OWL-14	20140111	23:00	2:00	6.4	5.8	2.4	3.9	3.9	1.00	1.00	1.00	3.5	1.00
OWL-15	20140114	0:00	12:00	3.7	2.9	0.9	16.1	19.9	1.00	1.00	1.00	15.3	1.00
OWL-16	20140114	12:45	15:45	4.1	3.6	1.2	2.6	3.1	1.00	1.00	1.00	1.8	1.00
OWL-17	20140119	0:00	12:00	-7.5	-7.8	1.3	1.9	3.3	0.91	0.84	0.81	1.8	0.88
OWL-18	20140119	18:00	0:00	-5.7	-6.5	1.9	1.9	2.6	0.81	0.61	0.69	2.3	0.69
OWL-19	20140120	0:00	2:00	-3.7	-4.1	3.1	0.9	1.0	0.67	0.50	0.52	0.8	0.51
OWL-20	20140120	2:00	12:00	-3.5	-3.9	2.1	3.8	4.8	0.78	0.66	0.66	3.5	0.72
OWL-21	20140127	18:00	0:00	-11.5	-12.3	1.7	4.0	6.6	0.86	0.65	0.75	2.8	0.69
OWL-22	20140128	0:00	6:00	-15.0	-15.6	0.4	6.3	10.3	1.00	0.95	0.94	5.6	0.96
OWL-23	20140128	6:00	12:00	-16.5	-17.3	0.9	8.0	12.1	0.94	0.83	0.86	8.1	0.88
a Event-averag	sed ambient tempe	srature											
<sup>b</sup> Event-averag	sed ice bulb tempe	stature											
<sup>c</sup> Event-averag	ged hotplate U												
<sup>1</sup> Liquid-equiv	'alent precipitation	1 amount no	nt corrected 1	or ineffic	cient catch	(UW valı	tes are co	vinputed v	with an $f_2$ deri	ved with the sect	ond of two m	ethods discusse	d in section 3.
e Event-averas	red snow narticle (	atch efficie	mev derived	V ouisu	12 and the	hotnlate 1	1.	•	5				

<sup>by</sup> Event-averaged snow particle catch efficiency derived using Y12 and the anenometer U<sup>f</sup> Event-averaged snow particle catch efficiency derived using X11 and hotplate U adjusted to 10 m AGL <sup>b</sup> Event-averaged snow particle catch efficiency derived using R11 and hotplate U (GLE and BTL) or anenometer U (GLE)

 $^{i}$  NA = not available  $^{j}$  NAP = not applicable



Table 5 – Precipitation Events





1	Table 6 – Summary of Fitt	ed Nu-Re Coefficie	ents					
2								
3								
1		Field Ca	alibrati	on				
4	(Nu - Re Coefficients)							
5		Hotplate/Field Site	γ	α	β			
		NCAR/BTL <sup>a</sup>	86.2	0.126	0.781			
6		UW/GLE <sup>b</sup>	49.1	0.130	0.771			
		UW/OWL °	45.6	0.172	0.713			
7								
0								
8								
9	<sup>a</sup> NCAR hotplate; measurem	ent interval 201201	18 23:	00 UTC	to 201	120119 5:00 U	JTC	
10	<sup>b</sup> UW hotplate; measurement	t interval 20120402	04:00	UTC to	0 20120	402 09:00 UT	Ċ	
11	<sup>c</sup> UW hotplate; measurement	t interval 20140107	18:00	UTC to	0 20140	108 08:00 UT	C	